

Turbulence

- Most practical Comb. is Turbulent.
 - large size
 - high velocity \rightarrow high Re.
- Desirable for mixing.
- Nonreacting jets \rightarrow jet flames.

Characteristics

- Why Does turb. happen?
 - Reynolds experiment: Pipe movie.
 - $Re \sim \frac{F_{\text{non}}}{F_{\text{visc}}} ;$ Flow instabilities are not damped by Viscosity.

- 2 key forces: mom, visc.
- Jet w/ Inc. Velocity (Re)



- As Ve inc., gradients (where visc. dissipates the exit momentum) increases, and gets very large.

- Diss. of mom $\sim T \cdot A = \mu \frac{du}{dx} \cdot A$

- Large area is not stable \rightarrow breaks apart

- Inc. S.A.

- Dec. (stable) Gradients.

$\left. \begin{array}{l} \\ \end{array} \right\} = \text{turbulence.}$

Kelvin-Helmholz movie.

- observations ??

Richardson 1922

Poem.

- ① Large eddies engulf fluid

- ② Small eddies grow to big eddies that decompose to small eddies

Turbulent Cascade

- ③ Large S.A. (interface)

- ④ Mixing rate

Scaling.

Kolmogorov theorems.

Turb. consists of eddies of Different Sizes.

- Consider Scales of turbulence.

- Integral Scale : large eddies : $L, u \rightarrow \tau = \frac{4}{3}u$

- Kolmogorov Scale : Small eddies : $\eta, u_m \rightarrow \tau_m = \eta/u_m$,

- $Re = \frac{Lu}{\nu}$

- $Re_\eta = \frac{\eta u_m}{\nu}$

- See Slides

$$\eta = \left(\frac{\nu^3}{\epsilon} \right)^{1/4}$$

$$u_m = (\epsilon \nu)^{1/4}$$

$$\tau_m = (\nu/\epsilon)^{1/2}$$

$$\epsilon = u^3/L$$

$$Re = Lu/\nu$$

$$\frac{L}{\eta} = Re^{3/4}$$

$$\frac{u}{u_m} = Re^{1/4}$$

$$\frac{\tau}{\tau_m} = Re^{1/2}$$

Basic modeling



Reynold Decomposition.

$$u = \bar{u} + u' \quad \bar{u} = \frac{1}{\Delta t} \int_0^{\Delta t} u(t) dt$$

$$\bar{u}' = 0$$

- Resolving all scales is expensive

$$\sim Re^3$$

- Solve average flow field. \rightarrow Reynolds Decomp.
- \rightarrow Need eqns.
- Apply averaging to N.S.

$$\nabla \cdot \vec{V} = 0$$

$$\frac{\partial \vec{V}}{\partial t} + \nabla \cdot \vec{V} \vec{V} = -\frac{1}{\rho} \nabla P - \frac{1}{\rho} \nabla \cdot \vec{F} + \vec{g}$$

2-D ~~BL~~ BL eq

$$\frac{\partial u}{\partial t} + \left(\frac{\partial}{\partial x} uu \right) \rightarrow \frac{\partial}{\partial y} uv = -\frac{\partial^2 u}{\partial y^2}$$

$$\frac{\partial}{\partial x} (\bar{u} + u')(\bar{u}u') = \frac{\partial}{\partial x} (\bar{u}\bar{u} + 2\bar{u}u' + u'u')$$

Avg:

$$\begin{aligned} & \frac{\partial}{\partial x} (\bar{u}\bar{u} + 2\bar{u}u' + u'u') \\ &= \frac{\partial}{\partial x} (\bar{u}\bar{u} + 2\bar{u}u' + \bar{u}'\bar{u}') \end{aligned}$$

other terms too.

$$\rightarrow \cancel{\frac{\partial u}{\partial t}} + \frac{\partial}{\partial x} \bar{u}\bar{u} + \frac{\partial}{\partial y} \bar{u}v = \cancel{\frac{\partial \bar{u}}{\partial x}} - \cancel{\frac{\partial}{\partial x} u'u'} - \cancel{\frac{\partial}{\partial y} \bar{u}'\bar{u}'} \quad \text{ignore}$$

Some eqn, w/ others AND New term ↑

New term is $-\frac{\partial}{\partial y} \overline{u'v'}$

$\circlearrowleft \overline{u'v'}$ is a viscous stress.

$$\frac{\tau}{\rho} = \frac{\mu}{\rho} \frac{du}{dx} \quad (\Rightarrow) \quad \frac{m^2}{s} \cdot \frac{m}{s} \cdot \frac{l}{m} = \frac{m^2}{s^2}$$

$$\overline{u'v'} \quad (=) \quad \frac{m^2}{s^2}$$

Model as a $\overline{u'v'} \sim \gamma_{turb} \cdot \frac{\partial \bar{u}}{\partial y}$

$$\gamma_{turb} \quad (\Rightarrow) \quad \frac{m^2}{s} \rightarrow \frac{l_m^2}{L} = l_m \left| \frac{d\bar{u}}{dy} \right|$$

$$\approx 0.1368 l_m (\bar{v}_{max} - \bar{v}_{min})$$

$$l_m = 0.075 \text{ Eqn } \% \text{ for jets}$$

Jet Soln.

Eqs same as laminar, but ω / μ_{turb} instead of μ

\rightarrow Same solution as as laminar.

but the form of μ_{turb} gives diff. behavior.

Turb.

$$\frac{\bar{u}_o}{\bar{u}_e} = 13.15 \frac{R}{x}$$

$$\frac{r_{1/2}}{x} = 0.05468$$

$\underbrace{\qquad\qquad\qquad}_{\text{indep. of Re!}}$

Laminar.

$$\frac{\bar{u}_o}{\bar{u}_e} = 0.378 \quad Re \cdot \frac{R}{x}$$

$$\frac{r_{1/2}}{x} = 2.97 / Re$$

$\underbrace{\qquad\qquad\qquad}_{\text{Dep. on Re!}}$